

# SHOCK ABSORBER

## Series SA1E

Cat No SA1E - 01 - 01 - A

### SHOCK ABSORBER - Stroke - 4, 6, 7, 10, 12, 15, 25 mm

#### Features

- Smooth and Quick deceleration characteristics
- Shock absorber resistance will be self-adjusted against the applied load
- Full threaded body design enhances heat dissipation & gives flexibility over fine adjustment
- High energy absorption characteristics & robust design gives longer life
- Higher usage frequency and smaller installation length



#### Applications

- Conveyor systems
- Factory automations
- Semi-Conductor Manufacturing
- Food processing equipment
- Metal Forming and Stamping equipment
- Medical Devices
- Automotive Manufacturing
- Glass forming equipment
- Pick & place robotics
- Material handling
- Packing machinery

#### Technical Specifications

Ambient Temperature	-10° to 80° C
Speed range	0.3 - 5.0 m/s
Reset time	≤ 0.3 Sec.
Material	Outer tube : Carbon Steel, Piston Rod : Carbon Steel (Hard Chrome plated), Seals : NBR

#### Performance Parameters

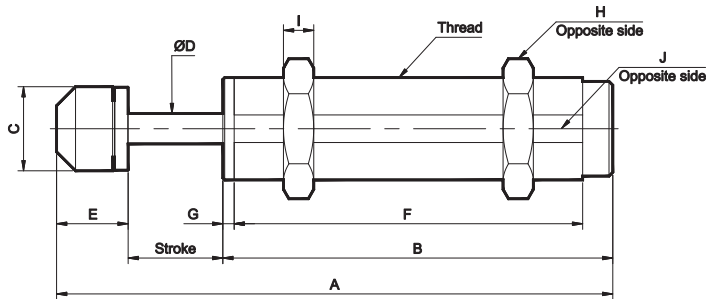
Stroke (mm)	Max. Nm/Cycle (Et)	Max. Nm/Hour (Etc)	Max. effective Mass Kg (Me)	Max. impact speed (v) m/s	Weight (g)	Ordering No (With Cap)	Ordering No (Without Cap)
4	0.5	720	3	0.3 - 1	4	SA1E004002XWC	SA1E004002YWC
6	3	7000	6	0.3 - 2.5	17	SA1E00603AXWC	SA1E00603AYWC
7	6	12400	12	0.3 - 3.5	28	SA1E007004XWC	SA1E007004YWC
10	12	22500	22	0.3 - 4	32	SA1E010005XWC	SA1E010005YWC
12	20	33000	40	0.3 - 5	70	SA1E01206AXWC	SA1E01206AYWC
15	59	38000	120	0.3 - 5	160	SA1E015008XWC	SA1E015008YWC
25	80	60000	180	0.3 - 5	295	SA1E025009XWC	SA1E025009YWC
25	147	72000	270	0.3 - 5	375	SA1E025010XWC	SA1E025010YWC

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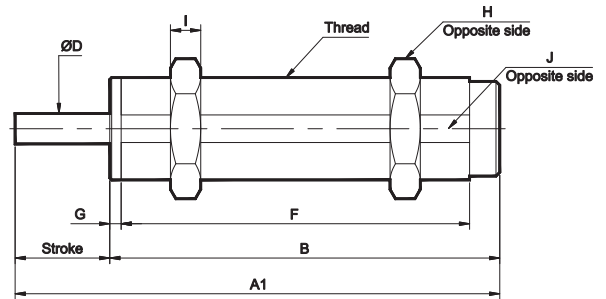
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**Shock Absorber with Cap**



**Shock Absorber without Cap**

Stroke	Thread	A	A1	B	C	D	E	F	G	H	I	J
4	M6 x 0.75	36.5	32.5	28.5	4.5	1.8	4	22.5	1	8	3	-
6	M8 x 1.0	55.2	46.6	40.6	6.6	2.9	8.6	33.6	2	11	3	-
7	M10 x 1.0	62.6	54	47	8.6	3	8.6	39	3	12.7	3	-
10	M12 x 1.0	71.3	62.5	52.5	10.3	3	8.8	44	3	14	4	-
12	M14 x 1.5	90.2	79	67	12	4	11.2	58	4	19	5	12.1
15	M20 x 1.5	103.3	88	73	17.8	6	15.3	62	4	26	7	18
25	M25 x 1.5	136	117	92	22	8	19	82	-	32	9	23
25	M27 x 1.5	143	124	99	22	8	19	86	5	32	6	25

### Shock Absorber recommendation chart for Air Cylinders

Cylinder I.D.	6	10	12	16	20	25	32	40	50	63	80
5 bar Cylinder propulsive force Kgf	1.4	3.9	5.7	10	15.7	24.5	40	62.8	98	155	251
SA1E004002XWC	•	•	•								
SA1E004002YWC											
SA1E00603AXWC		•	•	•							
SA1E00603AYWC											
SA1E007004XWC			•	•	•						
SA1E007004YWC											
SA1E010005XWC			•	•	•						
SA1E010005YWC											
SA1E01206AXWC			•	•	•						
SA1E01206AYWC											
SA1E015008XWC				•	•	•	•	•			
SA1E015008YWC											
SA1E025009XWC						•	•	•	•	•	•
SA1E025009YWC											
SA1E025010XWC						•	•	•	•	•	•
SA1E025010YWC											

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### Shock Absorber Selection

<b>E<sub>k</sub></b>	Kinetic energy	$E_k = mv^2 / 2$	(Nm)
<b>E<sub>D</sub></b>	Propelling energy	$E_D = F \times S$	(Nm)
<b>E<sub>T</sub></b>	Total energy per cycle	$E_T = E_k + E_D$	(Nm)
<b>E<sub>TC</sub></b>	Total energy per hour	$E_{TC} = E_T \times C$	(Nm/hr)
<b>Me</b>	Max. Effective mass	$Me = (2E_T) / V^2$	(kg)
<b>m</b>	Mass to slow down		(kg)
<b>*v</b>	Velocity or moving mass		(m/s)
<b>*v<sub>D</sub></b>	Impact velocity of shock absorber		(m/s)
<b>Fm</b>	Maximal impact force		(N)
<b>F</b>	Propelling force		(N)
<b>C</b>	Number of shocks per hour		(/hr)
<b>d</b>	Cylinder inner diameter		(mm)
<b>S</b>	Stroke		(m)
<b>A</b>	Width		(m)
<b>B</b>	Thickness		(m)

<b>T</b>	Driving torque	(Nm)
<b>I</b>	Moment of inertia	(kgm <sup>2</sup> )
<b>g</b>	Gravitational acceleration	(m/s <sup>2</sup> )
<b>H</b>	Height	(m)
<b>HM</b>	Arresting torque factor for motors (normally 2.5)	
<b>P</b>	Electric motor power	(W)
$\mu$	Friction coefficient	
<b>t</b>	Deceleration time	(s)
$\omega$	Angular velocity	(rad/s)
$\alpha$	Slope tilting angle	(rad)
$\theta$	Impact contact forwarding	(rad)
<b>P</b>	Actuation pressure	(bar)
<b>R</b>	Radius	(m)
<b>Rs</b>	Distance between shock absorber & rotational center	(m)

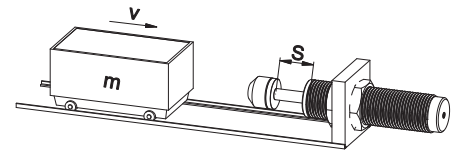
## 1. Horizontal Impact

### Usage conditions

$m = 2 \text{ Kg}$   
 $v = 0.7 \text{ m/s}$   
 $S = 0.004 \text{ m}$   
 $C = 1000/\text{hr}$

### Formula and Calculation

$E_k = mv^2 / 2 = (2 \times 0.7^2) / 2 = 0.5 \text{ Nm}$   
 $E_T = E_k = 0.5 \text{ Nm}$   
 $E_{TC} = E_T \times C = 0.5 \times 1000 = 500 \text{ Nm/hr}$   
 $Me = (2E_T) / v^2 = (2 \times 0.5) / 0.7^2 = 2 \text{ Kg}$



From the calculation result based on formula, it is recommended that **SA1E004002XWC** can be used.

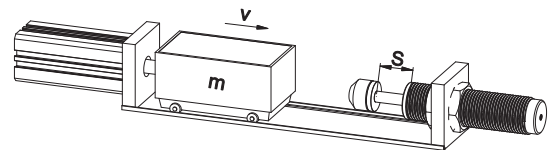
## 2. Horizontal Impact With Propulsive Force

### Usage conditions

$m = 2 \text{ Kg}$   
 $v = 1.2 \text{ m/s}$   
 $S = 0.006 \text{ m}$   
 $P = 3 \text{ bar}$   
 $D = 32 \text{ mm}$   
 $C = 2000/\text{hr}$

### Formula and Calculation

$E_k = mv^2 / 2 = (2 \times 1.2^2) / 2 = 1.44 \text{ Nm}$   
 $E_D = F \times S = 0.0785Pd^2 \times S = 0.0785 \times 3 \times 32^2 \times 0.006 = 1.45 \text{ Nm}$   
 $E_T = E_k + E_D = 1.44 + 1.45 = 2.89 \text{ Nm}$   
 $E_{TC} = E_T \times C = 2.89 \times 2000 = 5780 \text{ Nm/hr}$   
 $Me = 2E_T / v^2 = (2 \times 2.89) / 1.2^2 = 4.01 \text{ Kg}$



From the calculation result based on formula, it is recommended that **SA1E00603AXWC** can be used.

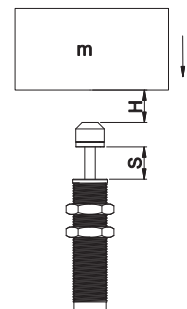
## 3. Free Fall Impact

### Usage conditions

$m = 10 \text{ Kg}$   
 $H = 0.05 \text{ m}$   
 $S = 0.007 \text{ m}$   
 $C = 2000/\text{hr}$

### Formula and Calculation

$v = \sqrt{2g \cdot H} = \sqrt{2 \times 9.81 \times 0.05} = 1 \text{ m/s}$   
 $E_k = mv^2 / 2 = (10 \times 1^2) / 2 = 5 \text{ Nm}$   
 $E_D = F \times S = mg \times S = 10 \times 9.81 \times 0.007 = 0.7 \text{ Nm}$   
 $E_T = E_k + E_D = 5 + 0.7 = 5.7 \text{ Nm}$   
 $E_{TC} = E_T \times C = 5.7 \times 2000 = 11400 \text{ Nm/hr}$   
 $Me = (2E_T) / v^2 = (2 \times 5.7) / 1^2 = 11.4 \text{ Kg}$



From the calculation result based on formula, it is recommended that **SA1E007004XWC** can be used.

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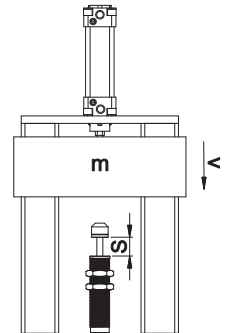
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### 4. Free Fall With Propelling Force

Usage conditions	Formula and Calculation
$m = 8 \text{ Kg}$	$E_k = mv^2 / 2 = (8 \times 1.0^2) / 2 = 4 \text{ Nm}$
$S = 0.01 \text{ m}$	$E_d = F \times S = (mg + 0.0785Pd^2) \times S$ $= (8 \times 9.81 + 0.0785 \times 4 \times 40^2) \times 0.01 = 5.8 \text{ Nm}$
$P = 4 \text{ bar}$	
$D = 40 \text{ mm}$	$E_T = E_k + E_d = 4 + 5.8 = 9.8 \text{ Nm}$
$C = 1800/\text{hr}$	$E_{TC} = E_T \times C = 9.8 \times 1800 = 17640 \text{ Nm/hr}$
$v = 1.0 \text{ m/s}$	$M_e = (2E_T) / v^2 = (2 \times 9.8) / 1.0^2 = 19.6 \text{ Kg}$

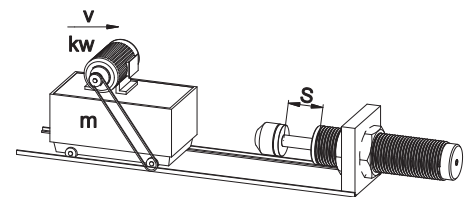
From the calculation result based on formula, it is recommended that **SA1E010005XWC** can be used.



### 5. Horizontal Impact With Motor Drive

Usage conditions	Formula and Calculation
$m = 5 \text{ Kg}$	$E_k = mv^2 / 2 = (5 \times 1.0^2) / 2 = 2.5 \text{ Nm}$
$v = 1.0 \text{ m/s}$	$E_d = F \times S = \{(P \times HM) / v\} \times S = \{(500 \times 2.5) / 1.0\} \times 0.012 = 15 \text{ Nm}$
$P = 0.5 \text{ kW}$	
$HM = 2.5$	$E_T = E_k + E_d = 2.5 + 15 = 17.5 \text{ Nm}$
$S = 0.012 \text{ m}$	$E_{TC} = E_T \times C = 17.5 \times 1500 = 26250 \text{ Nm/hr}$
$C = 1500/\text{hr}$	$M_e = (2E_T) / v^2 = (2 \times 17.5) / 1.0^2 = 35 \text{ Kg}$

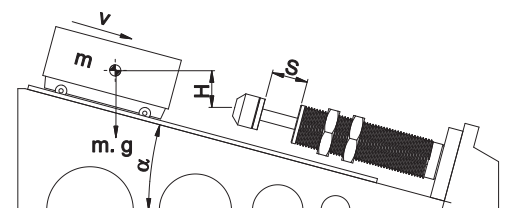
From the calculation result based on formula, it is recommended that **SA1E01206AXWC** can be used.



### 6. Tilted Impact

Usage conditions	Formula and Calculation
$m = 100 \text{ Kg}$	$v = \sqrt{2g.H} = \sqrt{2 \times 9.81 \times 0.1} = 1.4 \text{ m/s}$
$H = 0.1 \text{ m}$	$E_k = mv^2 / 2 = (100 \times 1.4^2) / 2 = 98 \text{ Nm}$
$S = 0.025 \text{ m}$	$E_d = F \times S = m \times g \times S \times \sin \alpha$ $= 100 \times 9.81 \times 0.025 \times \sin 30^\circ = 12.26 \text{ Nm}$
$\alpha = 30^\circ$	
$C = 600/\text{hr}$	$E_T = E_k + E_d = 98 + 12.26 = 110.26 \text{ Nm}$
	$E_{TC} = E_T \times C = 110.26 \times 600 = 66156 \text{ Nm/hr}$
	$M_e = (2E_T) / v^2 = (2 \times 110.26) / 1.4^2 = 112.5 \text{ Kg}$

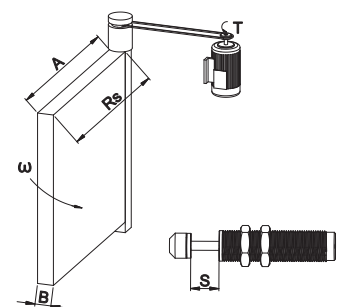
From the calculation result based on formula, it is recommended that **SA1E025010XWC** can be used.



### 7. Horizontal Rotational Door

Usage conditions	Formula and Calculation
$m = 100 \text{ Kg}$	$I = (m(4A^2 + B^2)) / 12 = (100(4 \times 1.0^2 + 0.05^2)) / 12 = 33.35 \text{ Kg m}^2$
$\omega = 2.0 \text{ rad/s}$	$E_k = (I\omega^2) / 2 = (33.35 \times 2.0^2) / 2 = 66.7 \text{ Nm}$
$T = 20 \text{ Nm}$	$\theta = S / R_s = 0.025 / 0.5 = 0.05 \text{ rad}$
$R_s = 0.5 \text{ m}$	$E_d = T \times \theta = 20 \times 0.05 = 1 \text{ Nm}$
$A = 1.0 \text{ m}$	$E_T = E_k + E_d = 66.7 + 1 = 67.7 \text{ Nm}$
$B = 0.05 \text{ m}$	$E_{TC} = E_T \times C = 67.7 \times 800 = 54160 \text{ Nm/hr}$
$S = 0.025 \text{ m}$	$v = \omega \times R_s = 2.0 \times 0.5 = 1 \text{ m/s}$
$C = 800/\text{hr}$	$M_e = (2E_T) / v^2 = (2 \times 67.7) / 1.0^2 = 135.4 \text{ Kg}$

From the calculation result based on formula, it is recommended that **SA1E025009XWC** can be used.



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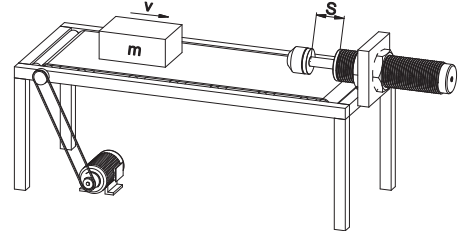
### 8. Horizontal Power Conveyor

**Usage conditions**

$m = 200 \text{ Kg}$   
 $v = 1.0 \text{ m/s}$   
 $\mu = 0.25$   
 $S = 0.025 \text{ m}$   
 $C = 600/\text{hr}$

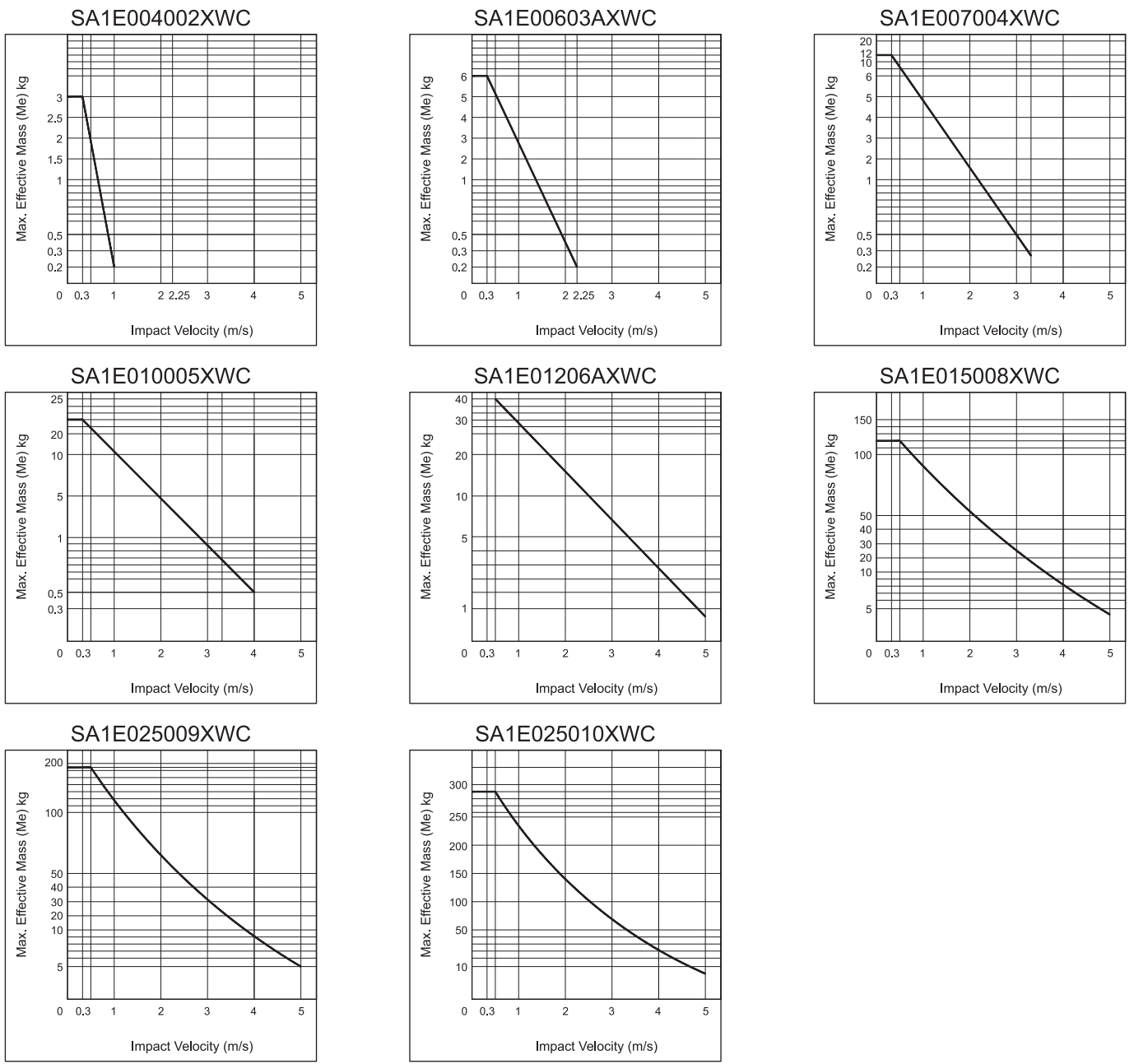
**Formula and Calculation**

$E_k = mv^2 / 2 = (200 \times 1.0^2) / 2 = 100 \text{ Nm}$   
 $E_D = F \times S = m \times g \times \mu \times S = 200 \times 9.81 \times 0.25 \times 0.025 = 12.26 \text{ Nm}$   
 $E_T = E_k + E_D = 100 + 12.26 = 112.26 \text{ Nm}$   
 $E_{TC} = E_T \times C = 112.26 \times 600 = 67356 \text{ Nm/hr}$   
 $M_e = (2E_T) / v^2 = (2 \times 112.26) / 1.0^2 = 224.52 \text{ Kg}$



From the calculation result based on formula, it is recommended that **SA1E025010XWC** can be used

### Flow Graph



### How to order

While ordering Shock Absorbers, mention the ordering number given in the corresponding table.

Subject to change